

# Hydrostatic Force on a Curved Surfaces

Henryk Kudela

## 1 Hydrostatic Force on a Curved Surface

On a curved surface the forces  $p\delta A$  on individual elements differ in direction, so a simple summation of them may not be made. Instead, the resultant forces in certain directions may be determined, and these forces may then be combined vectorially. It is simplest to calculate horizontal and vertical components of the total force.

### Horizontal component of hydrostatic force

Any curved surface may be projected on to a vertical plane. Take, for example, the curved surface illustrated in Fig. 1.

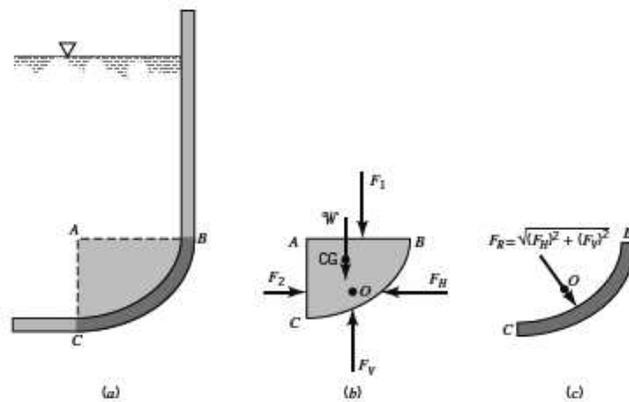


Figure 1: Hydrostatic force on a curved surface

Its projection on to the vertical plane shown is represented by the trace AC. Let  $F_x$  represent the component in this direction of the total force exerted by the fluid on the curved surface.  $F_x$  must act through the center of pressure of the vertical projection and is equal in magnitude to the force  $F$  on the fluid at the vertical plane.

In any given direction, therefore, the horizontal force on any surface equals the force on the projection of that surface on a vertical plane perpendicular to the given direction.

The line of action of the horizontal force on the curved surface is the same as that of the force on the vertical projection.

**Vertical component of hydrostatic force**

The vertical component of the force on a curved surface may be determined by considering the fluid enclosed by the curved surface and vertical projection lines extending to the free surface. Thus

$$F_H = F_2 = \rho g z_s \tag{1}$$

$$F_V = F_1 = \rho g V \tag{2}$$

where  $V$  is the volume of the liquid between the free surface liquid and solid curved surface. The magnitude of the resultant is obtained from the equation

$$F_R = \sqrt{F_H^2 + F_V^2} \tag{3}$$

**Example 1** A sector gate, of radius 4 m and length 5 m, controls the flow of water in a horizontal channel. For the (equilibrium) conditions shown in Fig. 2, determine the total thrust on the gate.

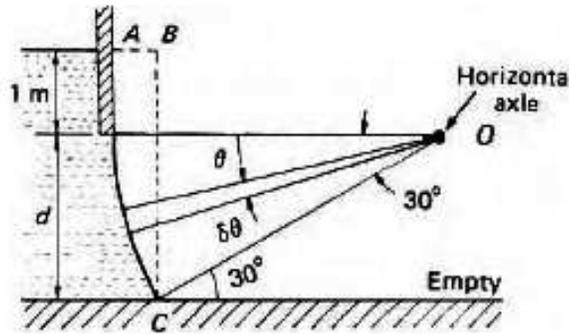


Figure 2: Hydrostatic force at a curved gate

**Solution.**

Since the curved surface of the gate is part of a cylinder, the water exerts no thrust along its length, so we consider the horizontal and vertical components in the plane of the diagram. The horizontal component is the thrust that would be exerted by the water on a vertical projection of the curved surface. The depth  $d$  of this projection is  $4 \sin 30^\circ = 2$  m, ( $R = 4$  m) and its centroid is  $1 + d/2 = 2$  m below the free surface. Therefore horizontal force  $F_H$  is equal

$$F_H = \rho g z_s A = 1000 \cdot 9.81 \cdot 2 \cdot (5 \cdot 2) = 1.962 \cdot 10^5 \text{ N}$$

Its line of action passes through the center of pressure of the vertical projection, that is, at a distance  $I_x / Az_s$  below the free surface, given by:

$$\frac{I_x}{Az_s} = \frac{I_s + Az_s^2}{Az_s} = \frac{bd^3}{12(bd)z_s} + z_s = \frac{2}{12 \cdot 2} + 2 + 1.167 \text{ m}$$



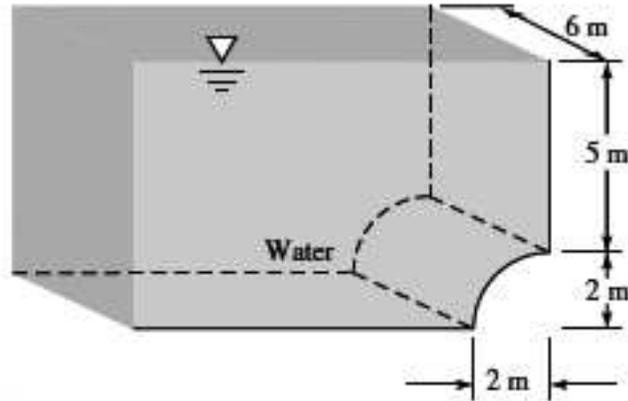


Figure 4: Hydrostatic force at the bottom of water tank

## 2 Buoyancy

Because the pressure in a fluid in equilibrium increases with depth, the fluid exerts a resultant upward force on any body wholly or partly immersed in it. This force is known as the **buoyancy**. This buoyancy force can be computed using the same principles used to compute hydrostatic forces on surfaces. The results are the two laws of buoyancy discovered by Archimedes in the third century B.C.:

1. a body immersed in a fluid experiences a vertical buoyant force equal to the weight of the fluid it displaces.
2. a floating body displaces its own weight in the fluid in which it floats.

So

$$\boxed{F_B = V_{body}\rho g} \quad (4)$$

In fig. 5 the vertical force exerted on an element of the body in the form of vertical prism of cross section  $\delta A$  is

$$\delta F = (p_2 - p_1)\delta A = \rho g h \delta A = \rho g \delta V$$

in which  $\delta V$  is the volume of the prism. Integrating over the complete body gives the formula (4).

Weighting an odd-shaped object suspended in two different fluids yields sufficient data to determine its weight  $W$ , volume  $V$ , specific weight  $\gamma = \rho g$  (or density  $\rho$ ) and specific gravity  $S_G = \frac{\rho}{\rho_{water}}$ . Figure 6 shows two free-body diagrams for the same object suspended and weighed in two fluids.  $F_1$  and  $F_2$  are the weight when submerged and  $\gamma_1 = \rho_1 g$  and  $\gamma_2 = \rho_2 g$  are the specific weights of the fluids.  $W$  and  $V$ , the weight and volume of the

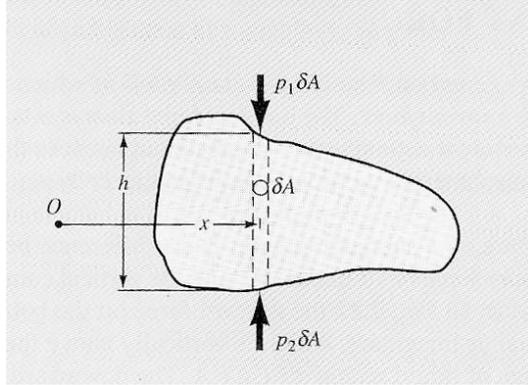


Figure 5: Vertical force componets on element of body

object, respectively, are to be found. The equations of equilibrium are written

$$F_1 + V \rho_1 g = W, \quad F_2 + V \rho_2 g = W$$

and solved

$$V = \frac{F_1 - F_2}{g(\rho_1 - \rho_2)}, \quad W = \frac{F_1 \rho_1 - F_2 \rho_2}{\rho_2 - \rho_1}$$

The hydrometr uses the principle of buoyant force to determine specific gravities of liquids.

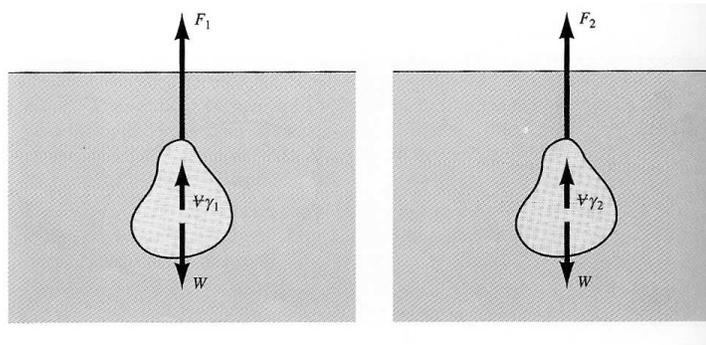


Figure 6: Free-body diagrams for body suspended in a fluid

Figure 7 shows a hydrometr in two liquids. It has a stem of prismatic cross section  $a$ . Considering the liquid on the left to be distilled water,  $S_G = 1.00$ , the hydrometr floats in equilibrium when

$$V_0 \rho g = W$$

in which  $V_0$  is the volume submerged,  $\rho g = \gamma$  is the specific weight of water, and  $W$  is the weight of the hydrometr. The position of the liquid surface is marked as 1.00 on the stem

to indicate unit specific gravity  $S_G$ . When the hydrometr is floated in another liquid the equation of equilibrium becomes

$$(V_0 - \Delta V)S_G\rho g = W$$

in which  $\Delta V = a\Delta h$ . Solving for  $\Delta h$  using above equations gives

$$\Delta h = \frac{V_0}{a} \frac{S_G - 1}{S_G}$$

from which the stem can be marked off to read specific gravities or densities.

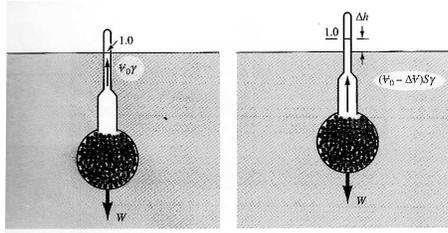


Figure 7: Hydrometr in water and in liquid of specific graviry  $S_G$ .

## References

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