### The 10<sup>th</sup> International Scientific Conference COMPUTER AIDED ENGINEERING

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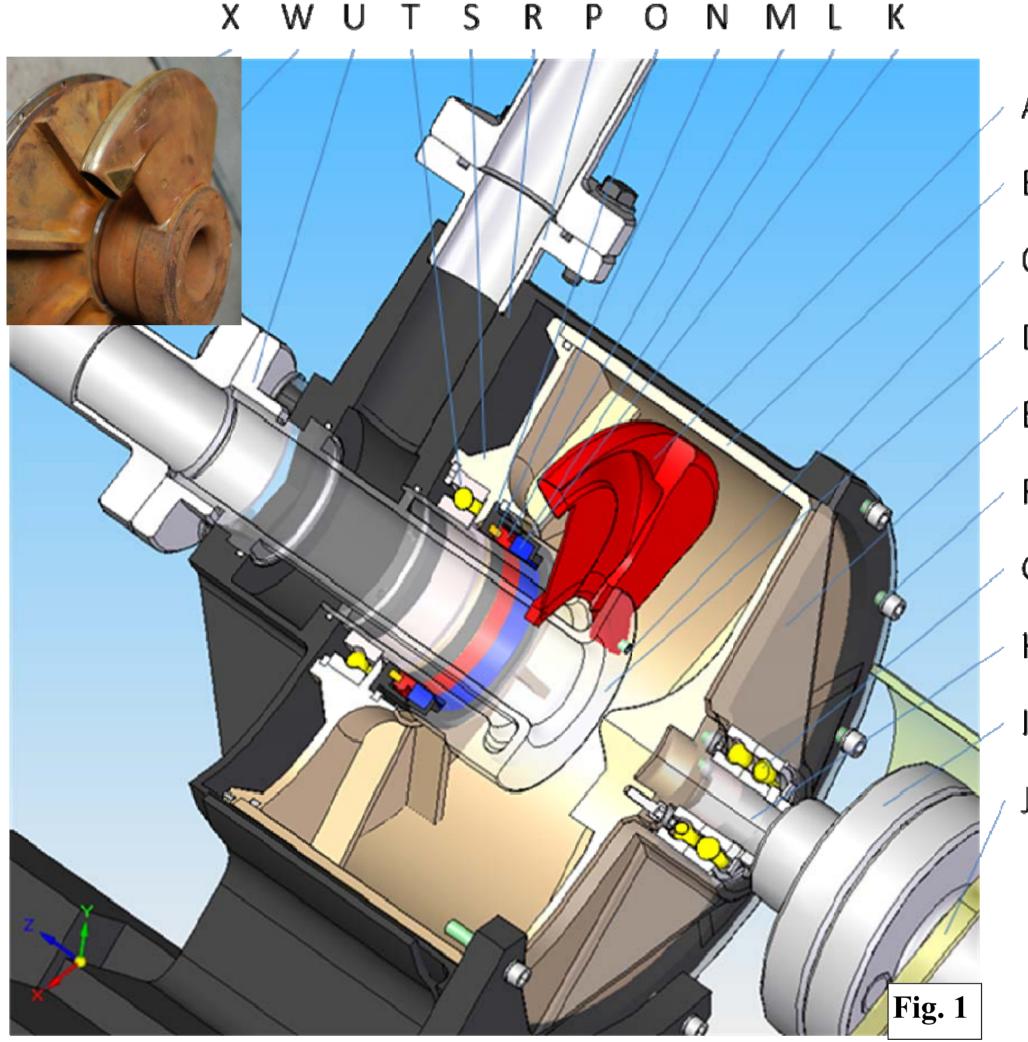


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## **INFLUENCE OF THE GEOMETRY DESIGN OF THE RIGID PITOT**

## **TUBE ENTRY ON THE TOTAL HEAD OF THE PITOT TUBE PUMP**

### **Geometry design**



### Stage 2

In the case of stage 2 central radial passage and axial-radial

pitot tube passage were changed. В

Four small walls (Y) forming

four pipes were removed leaving Fig. 5 D

- one canal, (l=50 mm long round
- discharge cylinder (Fig. 7)).
- All (circle-oval and triangle)
- the axial passage diffusers of
- pitot tube were connected at

# $\phi 45 mm$

 $H_{cz,A}=214,74$  m (cz.A÷F) as well as cz.3A (cz.1÷9)  $H_{cz.3A} = 203,31 \text{ m}.$ After doing the research of all radial passage 172,100 pitot tube by  $Q/Q_n=1$  were noticed for cz.DKKpBk;

 $\Delta H_{cz,DKKpBk}$ =6,928 m and cz.3A;  $\Delta H_{cz,3AKKpBk}$ =9,58 m.

40		-
	Hydraulic Losses, Hstrat=f(Q)	240 E cz3A, b/B=0,15, l/(pi*dsr)=0,2051; czA, b max/B=0,2347
35 -	Pitot tubes F÷A + cz7KKpZk	x cz6, b/B=0,144 I/(pi*dsr)=0,2055; czB, b max/B=0,2216 czF czE czD czC czB czA   230 cz5, b/B=0,119 I/(pi*dsr)=0,2062; czC, b max/B=0,200063 czF czE czD czC czB czA

### Results

total head by  $Q/Q_n=1$  was obtained for cz.A,

As the result of proper CFD modeling the highets net

Represented pump has nominal parameters as follows:

- Specific speed  $n_{\rm q} = 5,28$
- Total Head  $H_n = 148 \text{ m}$
- $Q_n = 15 \text{ m}^3/\text{h}$ Capacity
- $\eta_n = 33 \%$ Efficiency
- $Pw_n = 18 \text{ kW}$ • Power Consumption

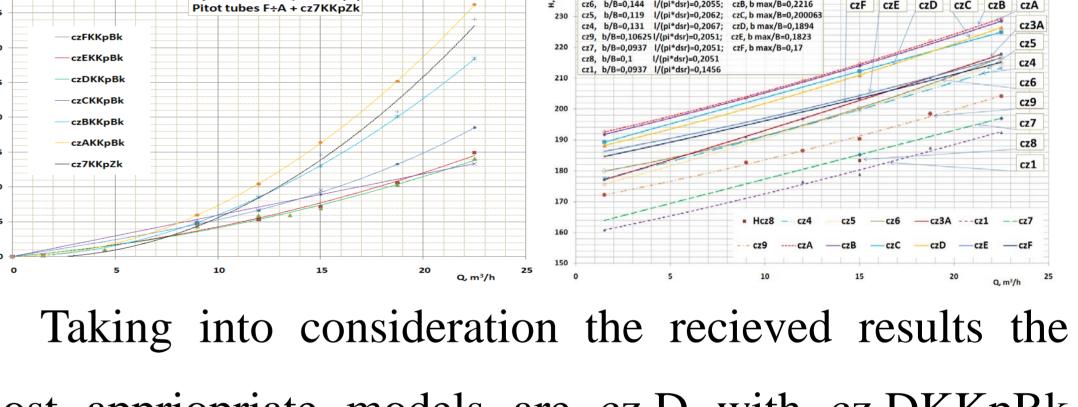
### Stage 1

Research contained 16 types of pick-up inlets into pitot tube models. Selected geometry parameters are demonstrated on the table 1 and in

right angle to discharge cylinder. Fig. 6

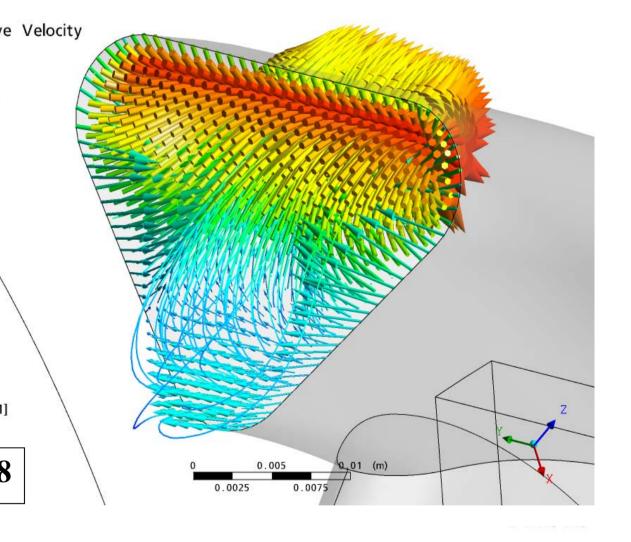
### **Turbulence modeling**

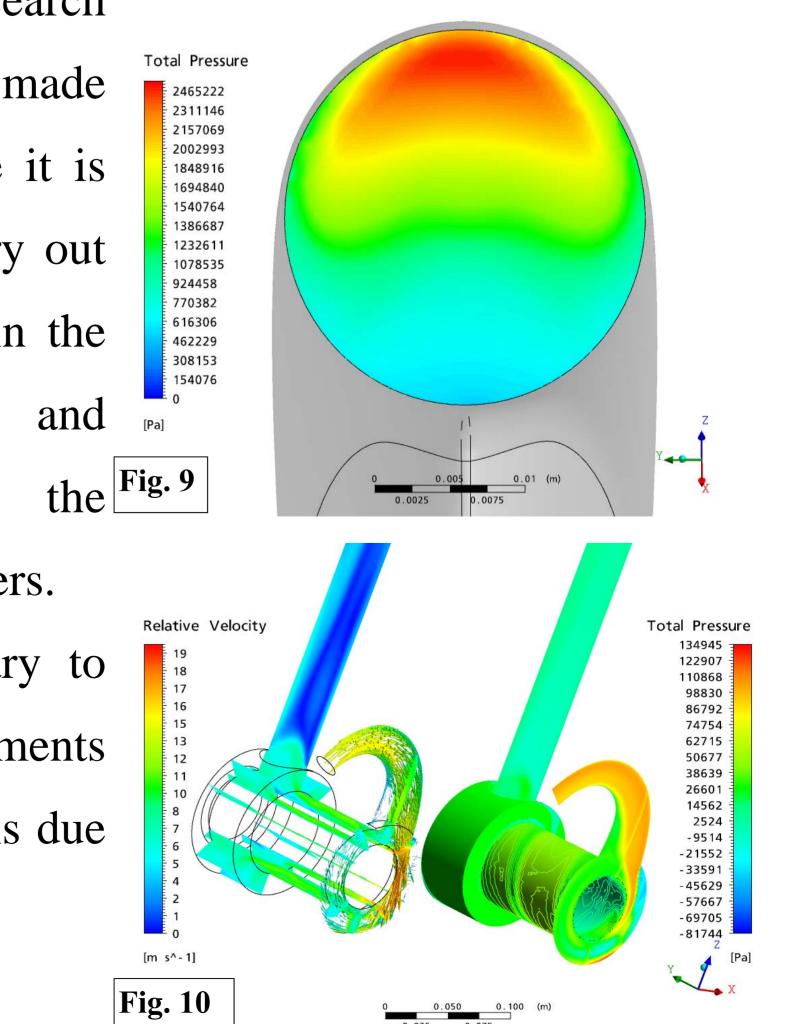
The complete formulation of Shear Stress Transport (SST) is described by equations (1), (2) and (3)•Turbulent Kinetic Energy  $\rho \frac{\partial \omega}{\partial t} + \rho u^{i} \frac{\partial \omega}{\partial x^{i}} = \frac{\partial}{\partial x^{j}} \left( \left( \mu + \frac{\mu_{t}}{\sigma_{\omega}} \right) \frac{\partial \omega}{\partial x^{j}} \right) + G_{\omega} - Y_{\omega} + D_{\omega}$ (1) •Specific Dissipation Rate  $\rho \frac{\partial k}{\partial t} + \rho u^{i} \frac{\partial k}{\partial x^{i}} = \frac{\partial}{\partial x^{j}} \left( \left( \mu + \frac{\mu_{t}}{\sigma_{k}} \right) \frac{\partial k}{\partial x^{j}} \right) + G_{k} - Y_{k} \quad (2)$ •Eddy Viscosity  $\mu_t = \rho \frac{\kappa}{-}$ (3)



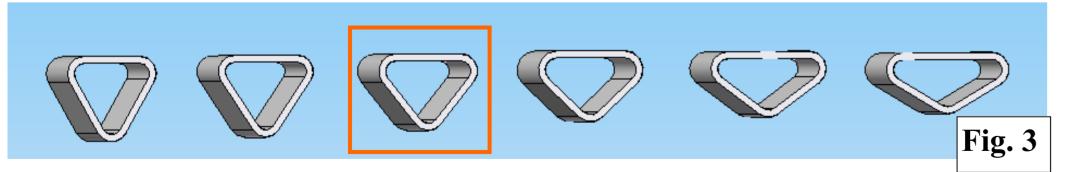
most appriopriate models are cz.D with cz.DKKpBk equals total head

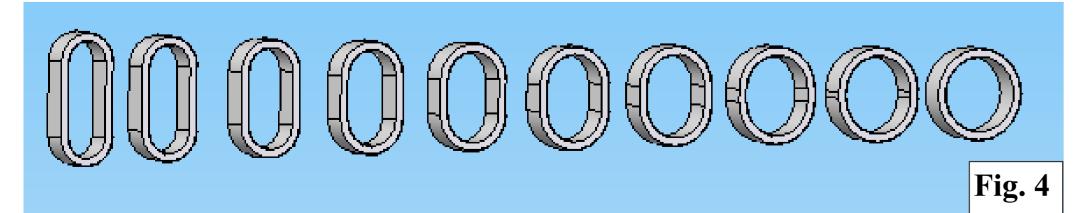
H<sub>cz.D</sub>=204,052 m, and second model cz.3A cz.3AKKpBk with achieving total head a m o u n t t o [m s^-1] Fig. 8  $H_{cz.3A} = 193,73$  m.





#### fig. 2 ÷ 4.





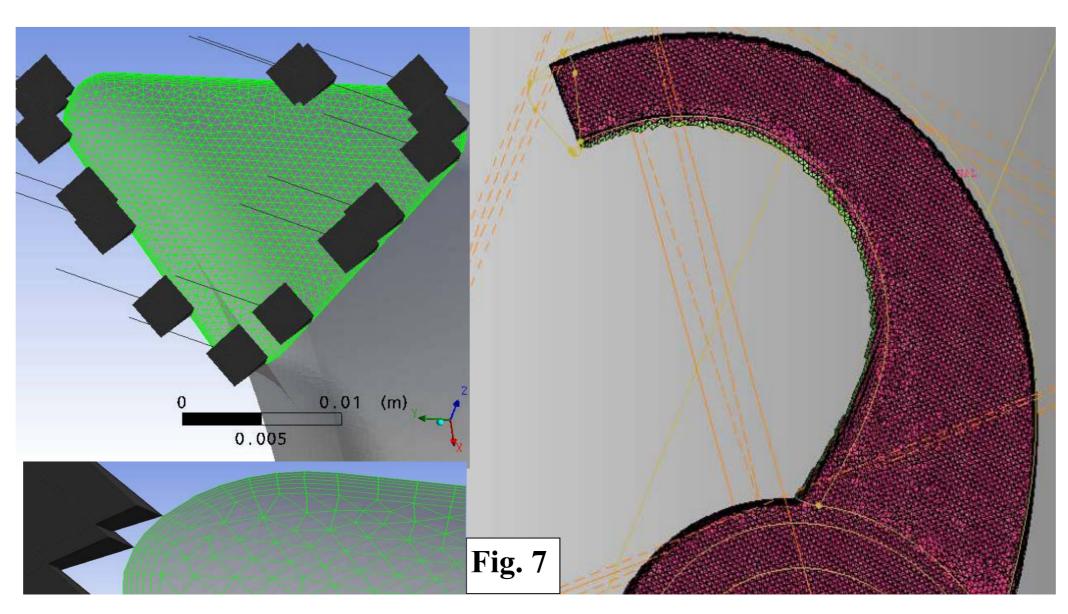
L.p.	Oznaczenie modelu czer- paka	b <sub>max</sub> mm	h <sub>c</sub> mm	h <sub>1</sub> mm	b mm	r mm	°	β °	b <sub>max</sub> / B -	b <sub>max</sub> /l <sub>p</sub> -
1.	cz. A	37,56	13,94	7,94	27,56	3	30	120	0,235	0,1639
2.	cz. B	35,46	15,2	9,2	25,46	3	36	108	0,222	0,1548
3.	cz. C	32,1	16,74	9,74	21,1	3	42,5	95	0,201	0,1401
4.	cz. D	30,31	18,37	12,37	20,31	3	50,5	79	0,189	0,1323
5.	cz. E	29,18	19,91	14,51	19,78	3	55,5	69	0,182	0,1274
6.	cz. F	27,31	20,99	17,31	14,99	3	60	60	0,171	0,1192

where:

**Fig. 2** 

- $G_k$  and  $G_{\omega}$  represent the generation of k and  $\omega$  $Y_k$  oraz  $Y_{\omega}$  represent the dissipation of k and  $\omega$
- $D_{\omega}$  is the cross-diffusion term

Descritization model is presented below:



simpler therefore it is necessary to carry out two pitot tubes in the physical models and

on

It is necessary to

elements

tests

nominal parameters.

all

from noble metals due

All the research

were made

models

do

make

to durability.